

Effective thermal conductivity of real two-phase systems using resistor model with ellipsoidal inclusions

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Abstract. A theoretical model has been developed for real two-phase system assuming linear flow of heat flux lines having ellipsoidal particles arranged in a three-dimensional cubic array. The arrangement has been divided into unit cells, each of which contains an ellipsoid. The resistor model has been applied to determine the effective thermal conductivity (ETC) of the unit cell. To take account of random packing of the phases, non-uniform shape of the particles and non-linear flow of heat flux lines in real systems, incorporating an empirical correction factor in place of physical porosity modifies an expression for ETC. An effort is made to correlate it in terms of the ratio of thermal conductivities of the constituents and the physical porosity. Theoretical expression so obtained has been tested on a large number of samples cited in the literature and found that the values predicted are quite close to the experimental results. Comparison of our model with different models cited in the literature has also been made.

Keywords. Real two-phase systems; effective thermal conductivity; correction term.

1. Introduction

The theoretical modelling for two-phase dissimilar systems of industrial importance is a challenging task for engineers and physicists, and of major interest to soil scientists and geologists. It is required because of increasing use of porous substances as insulating envelopes in solar ponds, non-conventional refrigerators, air conditioners and high temperature furnaces. The study of thermal parameters of these two-phase systems is also valuable for the explosive industry, the ceramics industry, nuclear reactors and in missile technology. High fluid porosity metal foams have long been used in design of aircraft wing structure in the aerospace industry, catalytic surfaces for chemical reactions, core structure for high strength panels, and containment matrix and burn rate heat exchanger for solid propellants. The ETC depends on various factors such as thermal conductivity, porosity, size of the particles and packing of the constituent phases. Accounting for all these factors in order to predict ETC is a complex affair. In the literature one finds several efforts (Babanov 1957; Brailsford and Major 1964; Pande *et al* 1984; Hadley 1986; Oshima and Watari 1989; Verma 1991) in which the situation has been simplified by assuming that the particles are of specific shape and arranged in particular geometries within the continuous phase.

The value of thermal conductivity in the solid–fluid composite is required in the numerical modelling of forced

convection through porous media (Poulikakos *et al* 1987). Hunt and Tien (1988) used an empirical stagnant conduction model developed by Tien and Vafai (1979) to define the effective thermal conductivity in the volume averaged homogeneous energy equation. Antohe *et al* (1996) also emphasized the use of an empirical phase symmetry conduction that was developed by Hsu *et al* (1994) to create a numerical model for the simulation of cooling micro heat exchangers. The origins of the phase-symmetry conduction model by Hsu *et al* (1994) are based upon the original work done by Zehner and Schlunder (1970) on packed beds of spheres. In a configuration with a low solid volume fraction and order of magnitude differences between the thermal conductivities of the two phases, the key in estimating the effective thermal conductivity is an accurate description of the geometry of the solid medium (Kaviany 1995). Zehner and Schlunder (1970) used this technique successfully for a packed bed of spheres. A recent advancement in the estimation of the effective thermal conductivity specifically for metallic foams saturated with a fluid utilizing a geometrical estimate was developed by Calmidi and Mahajan (1999) and Boomsma and Poulikakos (2001). For high porosity metal foams Calmidi and Mahajan (1999) presented a one-dimensional heat conduction model considering the porous medium to be formed of a two-dimensional array of hexagonal cells. Whereas Boomsma and Poulikakos (2001) proposed a three-dimensional model using metal foam structure in the form of tetrakaidecahedral cells with cubic nodes at the intersection of two nodes. Both the models involved a geometric parameter that was evaluated using the

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experimental data. Recently, Bhattacharya *et al* (2002) extended the analysis of Calmidi and Mahajan (1999) with a circular intersection, which results in a six-fold rotational symmetry.

In the present paper we have tried to fill the space arrangement of cells of equal size with the minimal surface energy and a theoretical model has been proposed to predict ETC of two-phase system with ellipsoidal inclusions. The arrangement has been divided into unit cells each of which contains an ellipsoid. The resistor model has been applied to determine the effective thermal conductivity (ETC) of the unit cell. However, in real systems, the packing and the shape of the particles are random. In order to incorporate varying individual geometries and non-linear flow of heat flux lines generated by the difference in thermal conductivities of constituent phases, a correction term in place of the physical porosity has been introduced. In the literature similar attempts have been made (Kampf and Karsten 1970; Koh and Fortini 1973; Peddicord 1976; Misra *et al* 1994; Singh *et al* 1995; Singh K J *et al* 1998), but for limited ETC ratios. Expressions for the porosity correction term have been obtained empirically by simulating experimental data reported in the literature. The present approach is simple and provides wider applicability to ellipsoidal model and enhances its ability to predict correctly the ETC of real two-phase system and systems having high ratios of thermal conductivities of their constituent phases.

2. Theory

We assume the following while analysing the problem: (i) the contact resistance between the solid and fluid phase is negligible, (ii) the mixture is homogeneous throughout and no transfer of heat occurs by way of convection or radiation and (iii) the heat flows along the x -axis and the flux lines remain parallel during the heat flow.

Let the grains of the solid phase be ellipsoidal having principal axes $2a$, $2c$ and $2a$ ($a < c$). Let these grains be located at the corners of a simple cube of side $2b$ each. Their distribution in two dimensions is shown in figure 1(a). The geometry of a unit cell is shown in figure 1(b).

Let the origin of coordinate axes be located at the centre of the ellipsoid. The unit cell can be divided into thin slices by planes perpendicular to the x -axis. Consider one such slice bounded by two planes at distances x and $x + dx$. The section shown in figure 1(c) is subdivided into four quadrants. One such section is shown in figure 1(d). This section is further divided by planes perpendicular to the z -axis. This results in the section of rectangular bars. One such bar is shown in figure 1(e). Let the length of the bar be b . Its cross-sectional area will be $dx dz$. The shaded portion of the element in figure 1(d) represents the solid phase and the non-shaded portion represents the fluid phase. This is supposed that the heat flux is incident normally on the cubic cell.

The volume fraction of the solid phase will be

$$(y dx dz)/(b dx dz) = y/b. \quad (1)$$

Similarly, the volume fraction of the fluid phase will be

$$\begin{aligned} & \{(b - y) dx dz\}/(b dx dz), \\ & = (1 - y/b). \end{aligned} \quad (2)$$

The terms (y/b) and $(1 - y/b)$ are equivalent to the one-dimensional porosity as used by Cheng and Vachon (1969). Considering various components as resistors one can take a combination of such resistors to predict ETC. As these elements form parallel resistors with respect to the direction of heat flow, therefore, using the resistor model the thermal conductivity of the bar will be

$$I' = I_1 (y/b) + I_2 (1 - y/b), \quad (3)$$

where I_1 and I_2 are the thermal conductivities of solid and fluid phases, respectively. With reference to figure 1(d), the thermal conductivity of the section will be

$$\begin{aligned} I'' &= \frac{(ab dx)}{b^2 dx} I'_{av} + \frac{(b - a) b dx}{b^2 dx} I_2, \\ I'' &= (a/b) I'_{av} + (1 - a/b) I_2, \end{aligned} \quad (4)$$

where

$$I'_{av} = (1/a) \int_0^a I' dz. \quad (5)$$

By combining (4) and (5), we get the following result

$$I'' = (1/b) \int_0^a I' dz + (1 - a/b) I_2. \quad (6)$$

With reference to figure 1(d), we have

Volume fraction of portions numbered 1

$$= (ab dx)/(b^2 dx) = (a/b), \quad (7)$$

Volume fraction of portions numbered 2

$$= \{(b^2 - ba) dx\}/(b^2 dx) = (1 - a/b). \quad (8)$$

These elements form equivalent series resistors perpendicular to the direction of heat flow, therefore, the effective thermal conductivity (I_e) of the unit cell can be written as

$$\frac{1}{I_e} = \frac{(a/b)}{I'_{av}} + \frac{(1 - a/b)}{I_2}. \quad (9)$$

Since I'' varies as x changes from 0 to a , therefore, on averaging

$$I''_{av} = (1/a) \int_0^a I'' dx. \quad (10)$$

By combining (6) and (10), we get the following result

$$I''_{av} = (1/a) \int_0^a [(1/b) \int_0^a I' dz + (1-a/b) I_2] dx. \quad (11)$$

By combining (3) and (11), we get the following result

$$I''_{av} = (1/a) \int_0^a [(1/b) \int_0^a \{I_1(y/b) + I_2(1-y/b)\} dz + (1-a/b) I_2] dx,$$

Therefore,

$$I''_{av} = \{(I_1 - I_2)/(ab^2)\} \int_0^a \int_0^a y dx dz + I_2. \quad (12)$$

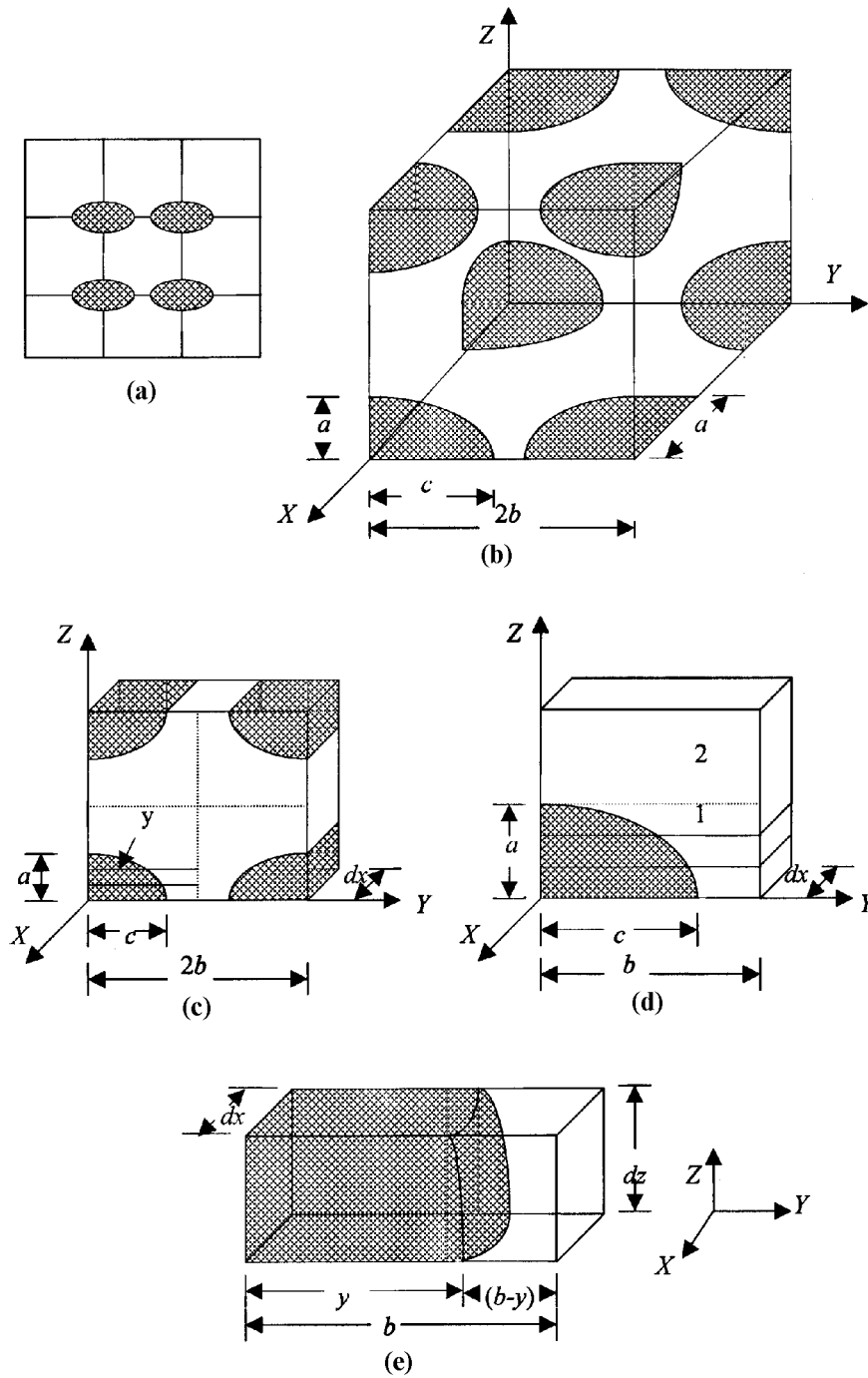


Figure 1. The resistor model for two-phase systems with ellipsoidal particles: (a) particles distribution in two dimensions, (b) geometry of a unit cell, (c) one section of unit cell, (d) one part of the section and (e) rectangular bar.

For an ellipsoidal particle we have

$$(x^2/a^2) + (y^2/c^2) + (z^2/a^2) = 1,$$

Therefore,

$$y = c \sqrt{1 - (x^2/a^2) - (z^2/a^2)}. \tag{13}$$

By combining (12) and (13), we get the following result

$$\begin{aligned} I_{av}'' &= \{(I_1 - I_2)/(ab^2)\} \int_0^a \int_0^a c \\ &\quad \sqrt{1 - (x^2/a^2) - (z^2/a^2)} dx dz + I_2, \\ &= \{(I_1 - I_2)/(ab^2)\} \{(pa^2c)/6\} + I_2, \end{aligned}$$

Therefore,

$$I_{av}'' = \{(I_1 - I_2)pac\}/(6b^2) + I_2. \tag{14}$$

By combining (9) and (14), we get the following result

$$\frac{1}{I_e} = \frac{(a/b)}{\{(I_1 - I_2)pac\}/(6b^2) + I_2} + \frac{(1-a/b)}{I_2},$$

Therefore,

$$I_e = \frac{I_2[(I_1 - I_2)\{(pac)/(6b^2)\} + I_2]}{[(1-a/b)(I_1 - I_2)\{(pac)/(6b^2)\} + I_2]}. \tag{15}$$

The unit cell contains one ellipsoid that lies inside. Therefore, fractional volume of the solid phase will be

$$f_1 = \frac{\{(4/3)(pa^2c)\}}{8b^3},$$

Therefore,

$$f_1 = \{(pa^2c)/(6b^3)\}. \tag{16}$$

Putting the limiting condition into (16), if $c = b$, we get the following result

$$f_1 = (p/6)(a^2/b^2),$$

Therefore,

$$(a/b) = \{\sqrt{(6/p)}\} f_1^{1/2}. \tag{17}$$

In this configuration the two ellipsoids are in contact with negligible contact resistance.

By combining (15) and (17), we get the following result

$$I_e = \frac{I_2[(I_1 - I_2)\{\sqrt{(p/6)}\} f_1^{1/2} + I_2]}{[(1 - \{\sqrt{(6/p)}\} f_1^{1/2})(I_1 - I_2)\{\sqrt{(p/6)}\} f_1^{1/2} + I_2]}. \tag{18}$$

For cubic packing of ellipsoidal inclusions the maximum value of the packing fraction will be < 0.52 (because $a < b$). So (18) is valid for $0 < f_1 < 0.52$ which is a low and medium dispersion case. In the limiting case, it can be seen that, when f_1 tends to 0, I_e approaches I_2 and when f_1

tends to 0.52, I_e leads to arithmetic mean of the phases. Noting that (18) is based on rigid geometry and it does not represent the true state of affairs of a real two-phase system. The ETC depends upon various characteristics of the system. The most prominent amongst them being the volume fraction and thermal conductivity of the constituent phases.

Thus, for practical utilization, we have to modify (18) by incorporating some correction term. Tareev (1975) has shown that, during the flow of electric flux from one dielectric to another dielectric medium, the deviation of flux lines in any medium depends upon the ratio of the dielectric constants of the two media. By the analogy we can have the concentration of thermal flux altered from its previous value as it passes through another medium and that the amount is a function of the thermal conductivities of the constituent phases. Such a deviation causes a zigzag path of flux lines in the bulk and also alters the density of flux lines in the constituent phases. The concentration of flux lines is greater in the phase of higher conductivity than it is in the phase of lower conductivity. If the flow of flux lines were linear then this porosity function would have been numerically equal to the physical porosity of the sample. In cases where curvature in the flow lines occur, the porosity function will not be equal to the physical porosity of the sample but it should be a function of the ratio of the thermal conductivities of the constituent phases as well as of the physical porosity of the sample. Considering random packing of phases, non-uniform shape of particles and the flow of heat flux lines not restricted to be parallel we here replace physical volume fraction of solid phase by porosity correction term F . F in general should be a function of the physical volume fraction of the solid phase and the ratio of the thermal conductivities of the constituent phases. Therefore, (18) may be written as

$$I_e = \frac{I_2[(I_1 - I_2)\{\sqrt{(p/6)}\} F^{1/2} + I_2]}{[(1 - \{\sqrt{(6/p)}\} F^{1/2})(I_1 - I_2)\{\sqrt{(p/6)}\} F^{1/2} + I_2]}. \tag{19}$$

Rearranging (19) we get

$$AF + BF^{1/2} + C = 0, \tag{20}$$

where

$$A = [Ie(I_1 - I_2)], B = [\{\sqrt{(p/6)}\}(I_1 - I_2)(I_2 - Ie)]$$

and $C = I_2(I_2 - Ie)$.

3. Results and discussion

Tables 1–2 cite experimental results of ETC and other data reported in literature. Without any correction term, (18) exhibits large deviations from the experimental results. This prompted the introduction of a correction in porosity.

The correction term introduced for each sample has been computed using (20) and plotted with $f_1^{1/2} \exp(I_2/I_1)$. Such plots of $f_1^{1/2} \exp(I_2/I_1)$ versus $F^{1/2}$ are shown in figures 2–4. It is observed from the figures that $F^{1/2}$ increases roughly

linearly with increasing $f_1^{1/2} \exp(I_2/I_1)$. We have used the curve fitting technique and found that the expression

$$F^{1/2} = C_1 f_1^{1/2} \exp(I_2/I_1) + C_2, \tag{21}$$

Table 1. Comparison of ETC values for two-phase systems using ellipsoidal particles (19). The thermal conductivity is in $W m^{-1} K^{-1}$.

Sl. No.	Type of the sample	f_1	I_1	I_2	I_e (expt)	I_e (theo.) by (19)	% Error
1	Cu/solder ^a	0.0124	398.0	78.1	79.8	80.918	1.4
2	Cu/solder ^a	0.0136	398.0	78.1	80.0	81.149	1.4
3	Cu/solder ^a	0.0507	398.0	78.1	85.2	87.591	2.8
4	Cu/solder ^a	0.0996	398.0	78.1	92.4	95.471	3.3
5	Cu/solder ^a	0.0195	398.0	78.1	80.8	82.249	1.8
6	Cu/solder ^a	0.0263	398.0	78.1	81.7	83.466	2.1
7	Cu/solder ^a	0.0286	398.0	78.1	82.0	83.869	2.3
8	Cu/solder ^a	0.1029	398.0	78.1	92.7	95.999	3.5
9	Cu/solder ^a	0.2377	398.0	78.1	115.4	118.878	3.0
10	Cu/solder ^a	0.0848	398.0	78.1	90.2	93.101	3.2
11	Cu/solder ^a	0.1586	398.0	78.1	102.0	105.068	3.0
12	Cu/solder ^a	0.2516	398.0	78.1	118.0	121.468	2.9
13	Cu/solder ^a	0.2894	398.0	78.1	125.0	128.821	3.0
14	Cu/solder ^a	0.291	398.0	78.1	125.0	129.143	3.3
15	Cellosize/flexol ^b	0.30	0.616	0.161	0.235	0.257	9.7
16	Water/oil solvent ^c	0.20	0.604	0.182	0.266	0.247	6.9
17	Cellosize/polypropylene glycol ^c	0.30	0.55	0.150	0.234	0.238	1.9
18	Water/mineral oil ^c	0.40	0.611	0.149	0.292	0.281	3.7
19	Selenium/polypropylene glycol ^d	0.40	5.192	0.14	0.422	0.363	13.9
20	Ti. Oxide/methylvinyl ^e	0.25	7.985	0.174	0.461	0.337	26.7
21	Graphite/water ^f	0.05	160.5	0.666	0.832	0.862	3.6
22	Graphite/water ^f	0.11	160.5	0.666	1.132	1.004	11.2
23	Graphite/water ^f	0.17	160.5	0.666	1.439	0.145	20.4
24	Selenium/polypropylene glycol ^d	0.10	5.208	0.14	0.18	0.197	9.7
25	Selenium/polypropylene glycol ^d	0.30	5.208	0.14	0.316	0.296	6.2
26	Selenium/polypropylene glycol ^d	0.40	5.208	0.14	0.423	0.363	14.1
27	Water/oil solvent ^c	0.20	0.605	0.182	0.267	0.247	7.2
28	Water/oil solvent ^c	0.40	0.607	0.173	0.312	0.316	1.3
29	Water/mineral oil ^c	0.20	0.611	0.149	0.234	0.208	11.0
30	Water/mineral oil ^c	0.40	0.611	0.149	0.293	0.281	4.1
31	Cellosize/F plasticizer ^b	0.10	0.551	0.166	0.190	0.196	3.2
32	Cellosize/F plasticizer ^b	0.10	0.577	0.190	0.21	0.222	6.1
33	Cellosize/F plasticizer ^b	0.30	0.467	0.135	0.18	0.212	18.1
34	Cellosize/F plasticizer ^b	0.30	0.551	0.166	0.236	0.259	10.1
35	Cellosize/F plasticizer ^b	0.30	0.577	0.190	0.256	0.292	14.4
36	Cellosize/polypropylene glycol ^b	0.10	0.551	0.150	0.182	0.178	1.8
37	Cellosize/polypropylene glycol ^b	0.10	0.577	0.154	0.180	0.183	2.0
38	Cellosize/polypropylene glycol ^b	0.30	0.467	0.110	0.157	0.179	14.1
39	Lead powder/Si rubber ^g	0.05	34.72	0.385	0.463	0.492	6.2
40	Lead powder/Si rubber ^g	0.16	34.72	0.385	0.651	0.637	2.0
41	Lead powder/Si rubber ^g	0.24	34.72	0.385	0.862	0.751	12.8
42	Bi powder/Si rubber ^g	0.05	8.33	0.385	0.433	0.468	8.2
43	Bi powder/Si rubber ^g	0.16	8.33	0.385	0.591	0.596	1.0
44	Bi powder/Si rubber ^g	0.24	8.33	0.385	0.734	0.696	5.1
45	ZnO/methyl vinyl ^e	0.15	23.1	0.1743	0.378	0.284	24.6
46	TiO/methyl vinyl ^e	0.25	7.81	0.174	0.462	0.337	27.0
47	Silica powder/dimethyl vinyl ^e	0.10	1.68	0.176	0.231	0.227	1.7
48	Silica powder/dimethyl vinyl ^e	0.15	1.68	0.174	0.252	0.246	2.09
49	Silica powder/dimethyl vinyl ^e	0.25	1.68	0.174	0.29	0.294	1.4
50	ZnO/synthetic rubber ^e	0.21	23.1	0.168	0.430	0.311	27.5
51	TiO/synthetic rubber ^e	0.18	7.81	0.168	0.359	0.283	21.0
Average deviation							7.8%

^aLee and Taylor (1976); ^bNahas and Couper (1966); ^cKnudsen and Wand (1958); ^dBaxley and Couper (1966); ^eRatcliffe (1962); ^fSugawara and Hamada (1970) and ^gCheng and Vachon (1969).

best fits the curve obtained in figures 2–4 where C_1 and C_2 are constants. These constants are different for different type of materials. The values of these constants for solid-air, emulsion, suspension, granular and solid–solid two-phase systems are 0.7282 and 0.0062, for Al-air system constants are 0.034 and 0.7111, for Al-water system constants are 0.5217 and 0.1535, for reticulated vitreous carbon (RVC)–air system constants are 0.491 and 0.5216 and for reticulated vitreous carbon (RVC)–water system constants are 0.6246 and 0.0604, respectively.

On putting (21) as the porosity correction term in (19) we have calculated values of ETC for a large number of samples reported in the literature. Tables 1–2 show a comparison of experimental results of ETC and calculated values from (19). The average deviation is 7.8% for solid-air, emulsion, suspension, granular and solid–solid two-phase systems shown in table 1 and for metal and nonmetal foams the average deviation is 6.2% shown in table 2, respectively. The constants C_1 and C_2 in (21) are different for metal

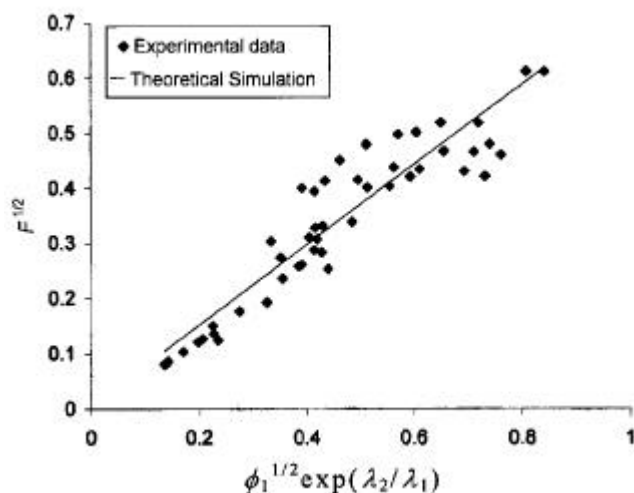


Figure 2. The variation of porosity correction term $F^{1/2}$ vs $f_1^{1/2} \exp(I_2/I_1)$.

Table 2. Comparison of ETC values for two-phase systems using ellipsoidal particles (19). The thermal conductivity is in $\text{W m}^{-1} \text{K}^{-1}$.

Sl. No.	Type of the sample	f_1	I_1	I_2	I_e (expt)	I_e (theo.) by (19)	% Error
1	Al/air ^h	0.029	218.0	0.026	2.7	2.720	0.7
2	Al/air ^h	0.054	218.0	0.026	4.6	3.916	14.8
3	Al/air ^h	0.095	218.0	0.026	6.7	8.454	26.2
4	Al/air ^h	0.051	218.0	0.026	3.9	3.742	4.0
5	Al/air ^h	0.091	218.0	0.026	6.7	7.684	14.7
6	Al/air ^h	0.022	218.0	0.026	2.2	2.455	11.6
7	Al/air ^h	0.051	218.0	0.026	4.0	3.742	6.4
8	Al/air ^h	0.094	218.0	0.026	6.9	8.249	19.5
9	Al/air ^h	0.028	218.0	0.026	2.5	2.681	7.2
10	Al/air ^h	0.048	218.0	0.026	3.9	3.579	8.2
11	Al/air ^h	0.063	218.0	0.026	4.5	4.511	0.2
12	Al/water ^h	0.029	218.0	0.615	3.7	3.562	3.7
13	Al/water ^h	0.054	218.0	0.615	5.4	4.813	10.8
14	Al/water ^h	0.095	218.0	0.615	7.65	8.432	10.2
15	Al/water ^h	0.051	218.0	0.615	4.8	4.641	3.3
16	Al/water ^h	0.091	218.0	0.615	7.6	7.917	4.1
17	Al/water ^h	0.022	218.0	0.615	3.05	3.262	6.9
18	Al/water ^h	0.051	218.0	0.615	4.95	4.641	6.2
19	Al/water ^h	0.094	218.0	0.615	7.65	8.298	8.5
20	Al/water ^h	0.028	218.0	0.615	3.3	3.518	6.6
21	Al/water ^h	0.048	218.0	0.615	4.75	4.477	5.7
22	Al/water ^h	0.063	218.0	0.615	5.35	5.381	0.5
23	RVC/air ^h	0.0336	8.5	0.026	0.164	0.162	1.1
24	RVC/air ^h	0.0276	8.5	0.026	0.15	0.151	0.8
25	RVC/air ^h	0.0385	8.5	0.026	0.17	0.171	0.9
26	RVC/air ^h	0.0319	8.5	0.026	0.16	0.159	0.6
27	RVC/water ^h	0.0336	8.5	0.615	0.73	0.731	0.3
28	RVC/water ^h	0.0276	8.5	0.615	0.722	0.720	0.2
29	RVC/water ^h	0.0385	8.5	0.615	0.743	0.741	0.3
30	RVC/water ^h	0.0319	8.5	0.615	0.727	0.728	0.2
Average deviation							6.2%

^hBhattacharya *et al* (2002).

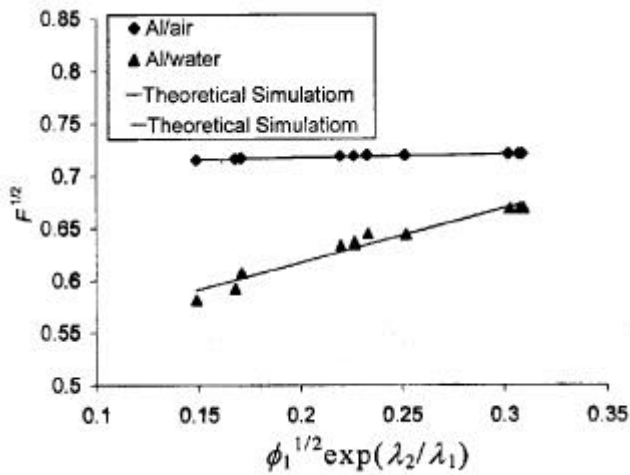


Figure 3. The variation of porosity correction term $F^{1/2}$ vs $f_1^{1/2} \exp(I_2/I_1)$.

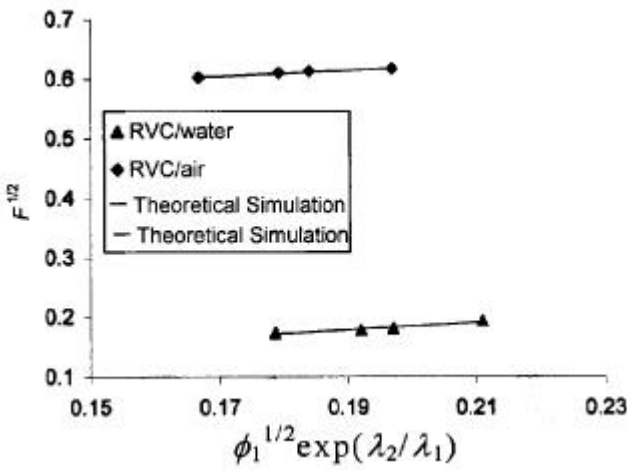


Figure 4. The variation of porosity correction term $F^{1/2}$ vs $f_1^{1/2} \exp(I_2/I_1)$.

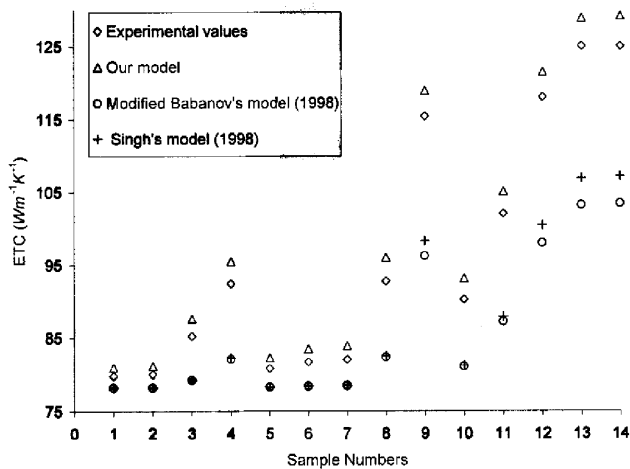


Figure 5. Comparison between experimental and theoretical values of ETC of the samples (sample nos 1–14, table 1).

foams. The reason is that a metal has very large thermal conductivity in the solid form, but even then, metal foams have exceptionally low ETC. In the solid form, other atoms of the metal bound each atom of the metal by a strong bond called metallic bond, which is responsible for the stiffness, hardness and high value of thermal conductivities for the metals. As we turn the metal to metal foam, the metallic bond no more exists in between the grains of the metal. Therefore, at the boundary of the grain a thin layer of insulating air is formed which reduces the metal to metal contact, thus, the overall ETC of the whole system reduces. The average percentage deviation from the experimental results for modified Babanov's model (Singh *et al* 1998) and Singh's (1998) model has been calculated and shown in figures 5–6. For foam-like materials we have used Boomsma's (2001) model and Bhattacharya's (2002) model for comparison of I_e as shown in figures 7–10. We have observed from these figures that when the volume fraction of the solid phase increases, ETC increases. We

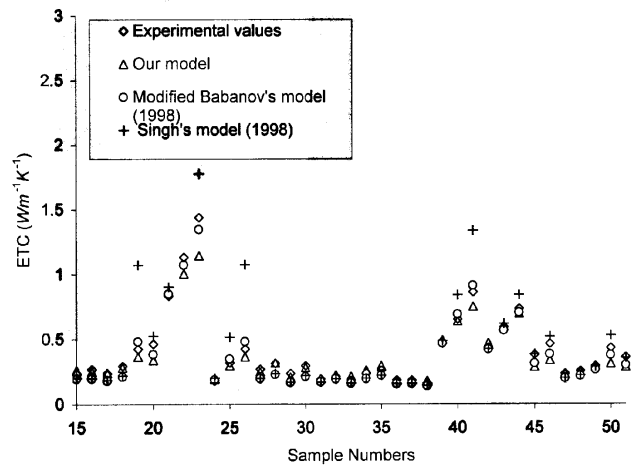


Figure 6. Comparison between experimental and theoretical values of ETC of the samples (sample nos 15–51, table 1).

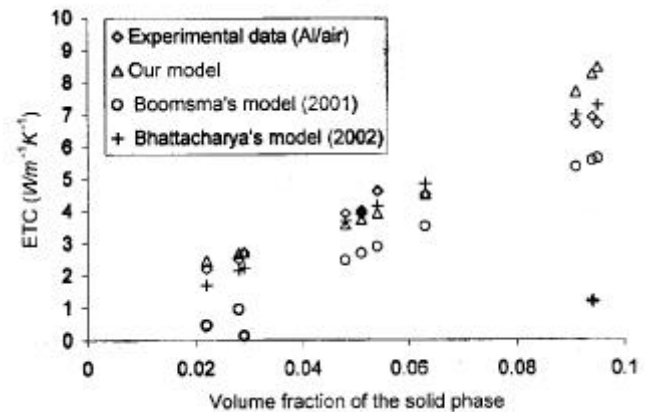


Figure 7. Comparison between experimental and theoretical values of ETC.

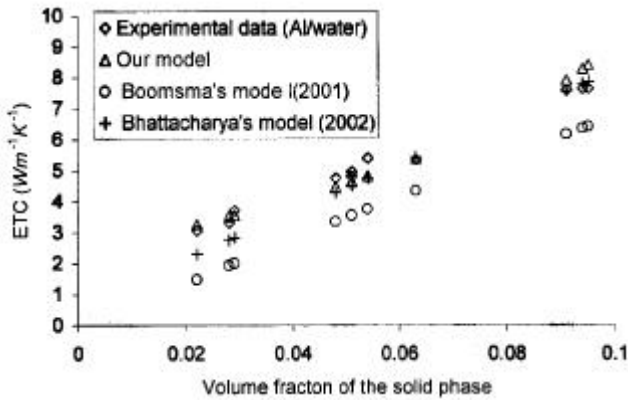


Figure 8. Comparison between experimental and theoretical values of ETC.

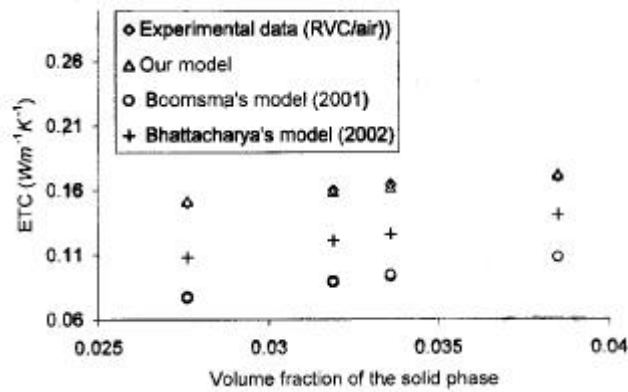


Figure 9. Comparison between experimental and theoretical values of ETC.

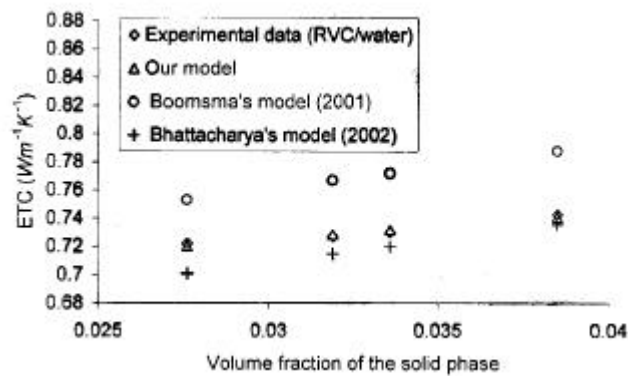


Figure 10. Comparison between experimental and theoretical values of ETC.

have also observed from figures 5–10 that for our model the average percentage deviation is better than other models.

4. Conclusions

(I) The model is capable of predicting ETCs close to the experimental values even for mixtures of higher conducti-

vity ratios and high porosities, whereas one may find that other models give higher deviations in those situations.

(II) This model enables one to avoid the introduction of sphericity or any other factor in the expression of ETC, making the model simple but powerful enough without compromising on the results.

(III) The proposed model for prediction of the ETC of two-phase systems holds not only for systems for which the ETC of the constituent phases are comparable, but also for systems having a high ETC ratio of the solid and fluid phases.

(IV) This model generalizes the work of Singh *et al* (1998), who treated a three-dimensional cubic array with spherical particles. The generalization to ellipsoidal particles is a useful one, since the ellipsoid can be used to model a variety of particle shapes, including discs and fibres in limiting cases.

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Appendix

The modified Babanov’s cubic particle model (Singh *et al* 1998) is

$$I_e = \frac{[I_2\{I_2 + F^{2/3}(I_1 - I_2)\}]}{[I_2 + F^{2/3}(I_1 - I_2)(1 - F^{1/3})]}, \tag{22}$$

where

$$F = [1 - \exp \{- (0.92)f_1^2 \ln (I_1/I_2)\}].$$

Singh’s (1998) spherical particle model is

$$I_e = \frac{[I_2\{I_2 + 0.8060F^{2/3}(I_1 - I_2)\}]}{[I_2 + F^{2/3}\{0.8060(I_1 - I_2)(1 - 1.2407F^{1/3})\}]}, \tag{23}$$

where

$$F = [1 - \exp \{- (0.92)f_1^2 \ln (I_1/I_2)\}].$$

The Boomsma’s (2001) model is

$$I_e = \frac{\sqrt{2}}{2[R_A + R_B + R_C + R_D]}, \tag{24}$$

where

$$R_A = 4F/[\{2e^2 + pF(1 - e)\}I_1 + \{4 - 2e^2 - pF(1 - e)\}I_2],$$

$$R_B = (e - 2F)^2/[(e - 2F)e^2I_1 + \{2e - 4F - (e - 2F)e^2\}I_2],$$

$$R_C = (\sqrt{2} - 2e)^2 / [\{2pF^2(1 - 2e\sqrt{2})I_1\} + 2\{\sqrt{2} - 2e - pF^2(1 - 2e\sqrt{2})\}I_2],$$

$$R_D = 2e/[e^2I_1 + (4 - e^2)I_2], F = \sqrt{[\sqrt{2}\{2 - (5/8)e^3\sqrt{2} - 2(1 - f_1)\}]/\{p(3 - 4e\sqrt{2} - e)\}},$$

and

$$e = 0.339.$$

Bhattacharya's (2002) model is

$$I_e = F\{f_1I_1 + (1 - f_1)I_2\} + \frac{(1 - F)}{\{f_1/I_1 + (1 - f_1)I_2\}}, \quad (25)$$

where

$$F = 0.35.$$

Symbols involved in the formulae (22)–(25) have the same meaning as in the previous part in the paper.

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